

# ME 314 - Engineering Design : Mechanical Components

## Lecture 3

Note Title

### 3.9 Beam Loading (Shear Force & Bending Moment)

A beam is an element that carries loads transverse to its long axis and may carry loads in the axial direction as well.

#### Types of Support

Simply Supported Beam:

Cantilever Beam:

Overhung Beam:

Indeterminate Beam:  
(has more supports than needed)

#### Types of Load

#### Sign Convention

#### Relation between $q(x)$ , $V(x)$ , and $M(x)$

Shear force,  $V$ , the bending moment,  $M$ , and the loading function,  $q$ , are related by

Given  $q(x)$ , integrate between, say,  $x_A$  and  $x_B$ :

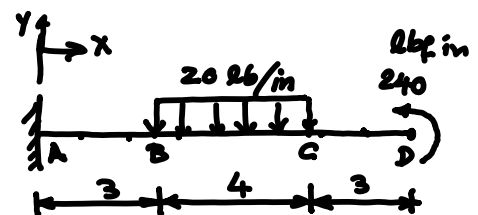
**Change in shear force from A to B equals the area of the loading diagram**

**Change in moment from A to B equals the area of the shear force diagram between  $x_A$  and  $x_B$ .**

### Singularity Functions

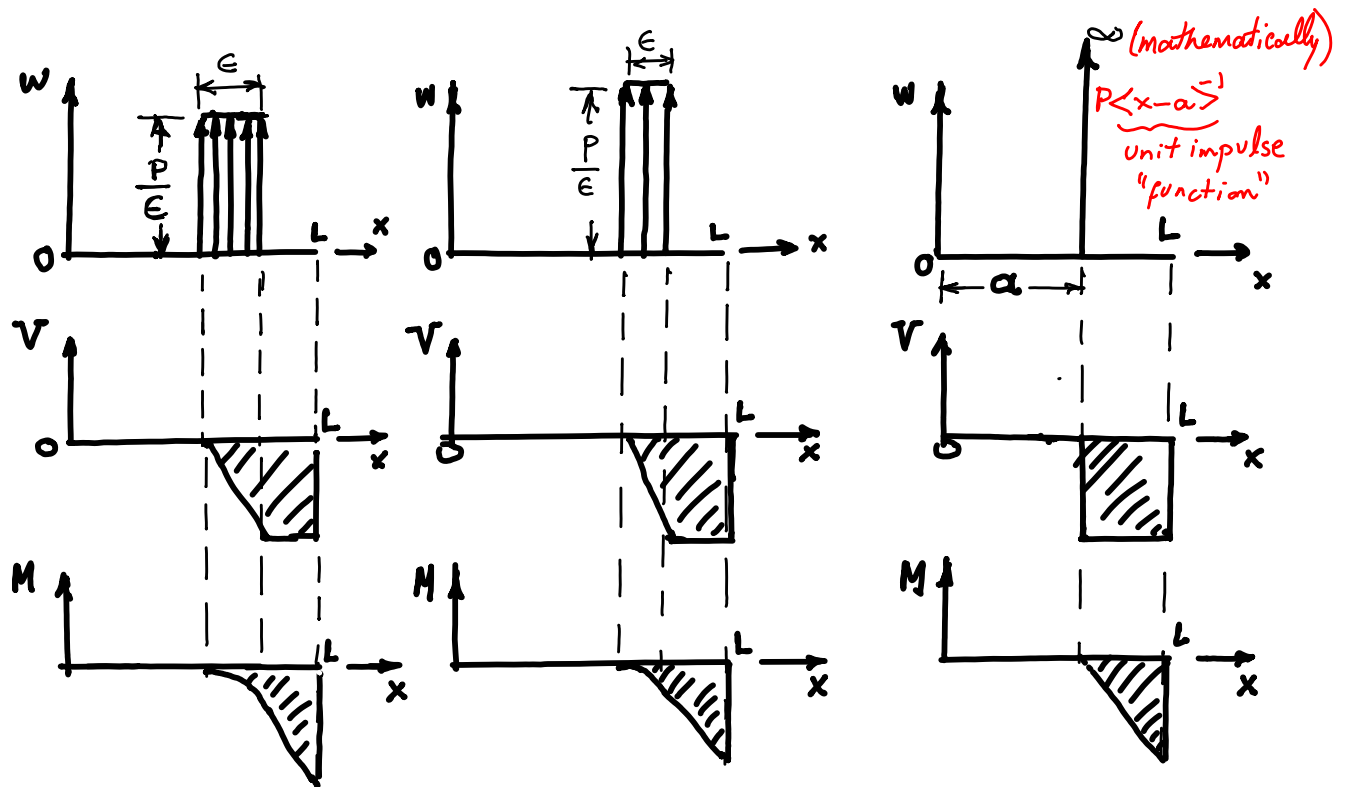
These are special class of functions that are useful for dealing with discrete functions such as point loads or segments of distributed loads.

For a problem such as the one shown in the figure, the usual approach of making cuts and finding the shear and moment expressions would necessitate that shear and moment expressions be constructed for each of the three intervals  $0 \leq x \leq 3$ , and  $3 \leq x \leq 7$ . Singularity functions permit us to write a single expression for the moment that is valid over the entire length of the beam. This is particularly helpful in calculating the bending deflection of a beam by integrating the expression for moment twice according to the beam equation  $EI y''(x) = M$ . This procedure will be reviewed later in Section 4.10.



## Concentrated Loads

Consider a distributed load  $w$  that acts over a relatively small segment  $\epsilon$  of a beam as shown.

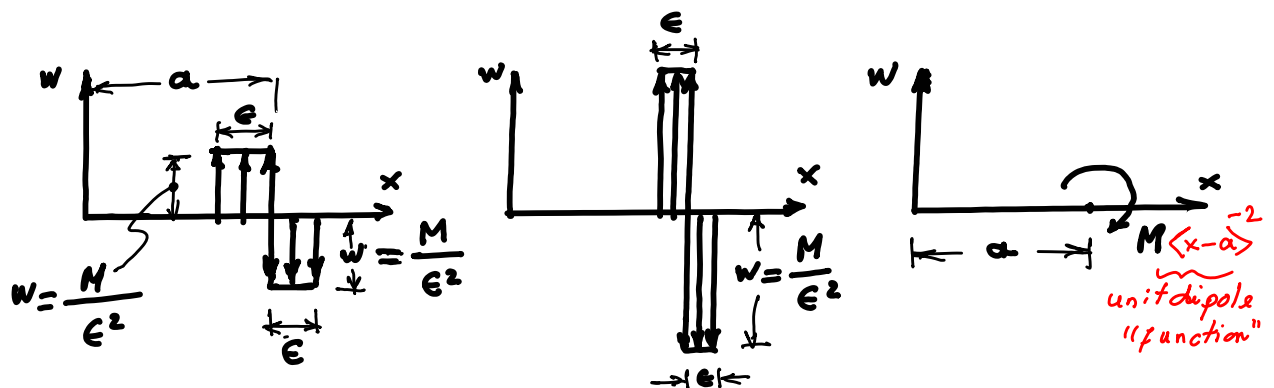


The singularity function is not actually a "function," rather only an indication that something rather unusual is being modeled, namely, a finite load  $P$  applied over an extremely small distance  $\epsilon$ .

Other symbols used for  $\langle x-a \rangle^{-1}$  are  $\delta(x,a)$  and  $\delta(x-a)$ . These symbols are often referred to as the **Dirac delta function** or the **unit impulse function**.

### Concentrated Moments

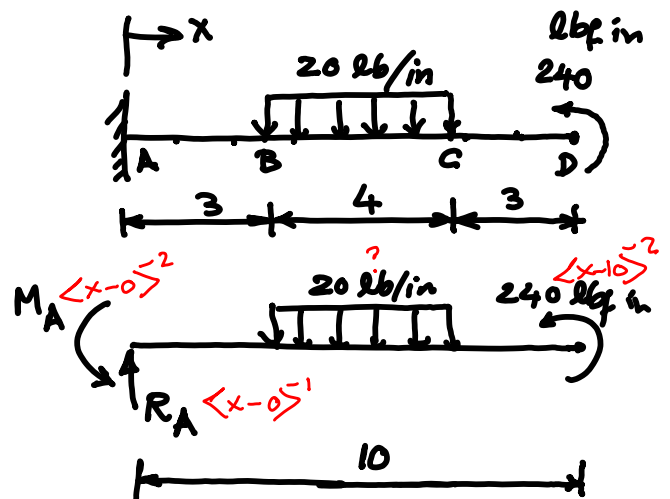
Now consider the loading shown in figure, where two adjacent segments of the beam are subjected to loads  $w\epsilon$  that have opposite sense.



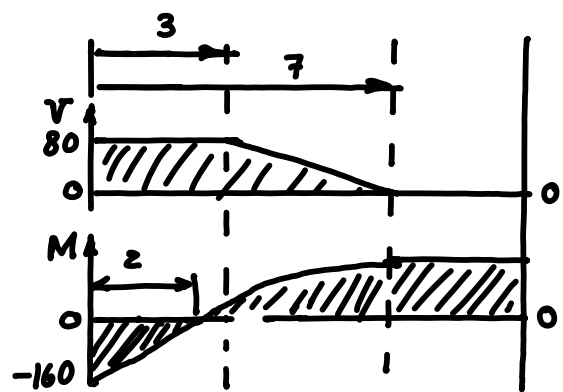
## Singularity Functions

Name	Definition	Graph	Integral
Unit dipole function at point $a$	$\langle x-a \rangle^{-2} = \begin{cases} 0 & \text{if } x \neq a \\ +\infty & \text{if } x = a \end{cases}$		$\int \langle x-a \rangle^{-2} dx = \langle x-a \rangle^{-1}$
Unit impulse function at point $a$	$\langle x-a \rangle^{-1} = \begin{cases} 0 & \text{if } x \neq a \\ +\infty & \text{if } x = a \end{cases}$		$\int \langle x-a \rangle^{-1} dx = \langle x-a \rangle^0$
Unit step function at point $a$	$\langle x-a \rangle^0 = \begin{cases} 0 & \text{if } x < a \\ 1 & \text{if } x \geq a \end{cases}$		$\int \langle x-a \rangle^0 dx = \langle x-a \rangle^1$
Ramp function at point $a$	$\langle x-a \rangle^1 = \begin{cases} 0 & \text{if } x < a \\ x-a & \text{if } x \geq a \end{cases}$		$\int \langle x-a \rangle^1 dx = \frac{1}{2} \langle x-a \rangle^2$
Parabolic function at point $a$	$\langle x-a \rangle^2 = \begin{cases} 0 & \text{if } x < a \\ (x-a)^2 & \text{if } x \geq a \end{cases}$		$\int \langle x-a \rangle^2 dx = \frac{1}{3} \langle x-a \rangle^3$

**Example:** A cantilever beam is loaded as shown. Derive the shear-force and bending-moment relations, and the support reactions.

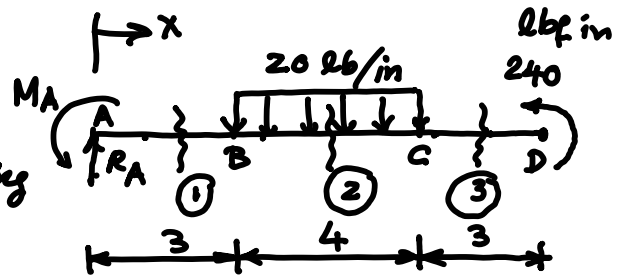


**Caution:** Note that both  $M_A$  and the 240 lb-in moments are counterclockwise and negative singularity functions; however, by the convention shown on page 4 above,  $M_A$  and the 240 lb-in moments are negative and positive moments, respectively, when we draw the moment diagram.



Method of Cuts:

We review this method by solving the above example.



Reactions :  $\sum M_A = 0$        $M_A - (20)(4)(5) + 240 = 0$

$M_A = 160 \text{ lb}\cdot\text{in}$

$\sum F_y = 0$        $R_A - (20)(4) = 0$  ,  $R_A = 80 \text{ lb}$

cut (1)  $\sum M_{cut} = 0$  :  $+M - 80x + 160 = 0$   
 $\sum F = 0$  :  $-V + 80 = 0$

The free body diagram for cut (1) shows a segment of length  $x$  from A. It has a counter-clockwise moment  $M$  at the cut, an upward reaction of  $80$  at A, and a downward shear force  $V$  at the cut. The coordinate  $x$  is indicated at the bottom.

$V = 80$  ,  $M = -160 + 80x$  ,  $0 \leq x \leq 3$

cut (2)  $\sum M_{cut} = 0$  :  $+M + 160 - 80x + 20\left(\frac{x-3}{2}\right)^2 = 0$   
 $\sum F = 0$  :  $-V + 80 - 20(x-3) = 0$

The free body diagram for cut (2) shows a segment of length  $x$  from A. It includes the reaction  $80$  at A, the moment  $M$  at the cut, and the distributed load from B to the cut. The shear force  $V$  acts downwards at the cut. The coordinate  $x$  is indicated at the bottom.

$V = 140 - 2x$  ,  $M = -160 + 80x - 10(x-3)^2$  ,  $3 \leq x \leq 7$

cut (3)  $\sum M_{cut} = 0$  :  $-M + 240 = 0$   
 $\sum F = 0$  :  $+V = 0$

The free body diagram for cut (3) shows a segment of length  $x$  from A. It includes the reaction  $80$  at A, the moment  $M$  at the cut, and the distributed load from B to C. The shear force  $V$  acts upwards at the cut. The coordinate  $x$  is indicated at the bottom.

$V = 0$  ,  $M = 240$  ,  $7 \leq x \leq 10$

### 3.8 Impact Loading

Please study this section on your own.